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AN INTEGRAL-EQUATION SOLUTION FOR A BOUNDED ELASTIC BODY WITH AN EDGE CRACK: MODE I DEFORMATIONS

METALS BEHAVIOR BRANCH METALS AND CERAMICS DIVISION

JULY 1978

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FOREWORD

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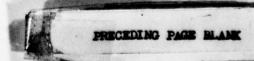
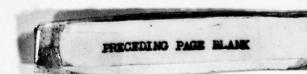


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SECTION I

INTRODUCTION

Integral equation methods have been shown to be effective means of formulating crack problems in linear elastic fracture mechanics. For some crack problems in unbounded media, explicit evaluation of quadrature form solutions are possible; however, for problems in finite media, numerical evaluation of integral as well as other formulations has been required.

In a previous paper [1], a quadrature form solution to a general, bounded, plane problem with an enclosed crack was formulated and a numerical solution technique developed. A set of coupled equations, involving integrals over the outer boundary and the line of the crack, were obtained by the simultaneous solution of a crack problem in an unbounded medium - the perturbed problem - and a crack problem in the unflawed medium - the equable problem - having the same bounding contour as the problem to be solved - the original problem. The perturbed problem, whose solution was expressed in quadrature form, corresponded to that of a crack in an infinite medium with prescribed crack surface tractions. The equable problem corresponded to that in which boundary tractions and/or displacements were prescribed on the outer boundary of the original, unflawed medium and was formulated by the direct potential or boundary integral equation (BIE) method. In this report, an analogous perturbed problem, viz., that of a

Regarded as prescribed insofar as the individual problem formulation is concerned, but remain to be determined by the simultaneous solution process.

semi-infinite crack in an infinite medium, is employed to solve boundary value problems of a plane body with an edge crack.

SECTION II

FORMULATION

Let the linearly elastic, isotropic body occupy the region bounded by the piecewise smooth curve C. Locate the x_1 (i = 1,2) coordinate system at the crack tip such that the undeformed crack surfaces are coincident with the line segment $0 \le x_1 \le a$, $x_2 = 0$ as shown in Fig. 1. Denote an arbitrary field point by $x = (x_1, x_2)$ and the field variables – displacement, traction and stress components – at x by $u_1(x)^2$, $t_1(x)$ and $\sigma_{ij}(x)$, respectively. In addition, let the field point x also have the complex rectilinear and polar representations $x = x_1 + ix_2 = re^{i\Theta}$.

1. Perturbed Problem

For mode I deformations, the perturbed problem corresponds to that of a semi-infinite crack in the infinite plane with a prescribed distribution of stress on a portion of the crack to be denoted by:

$$\sigma_{22}(x_1,0) = \begin{cases} \sigma(x_1), & 0 \le x_1 \le a \\ 0, & x_1 > a \end{cases}$$
 (1)

The solution to this problem is to be constructed by superimposing the results of a continuous distribution of crack opening point forces on the upper and lower crack surfaces as shown in Fig. 2.

For general mode I problems, solutions may be generated by the complex stress potential function Z(*) of Westergaard [2] through the relations

²Latin subindices have the range (1,2) and denote Cartesian components relative to the x_i reference frame. Repeated subindices imply summation and partial differentation is indicated by a comma between subindices.

$$\sigma_{11}(x) = \text{Re}[Z] - x_2 \text{Im}[Z']$$
 (2a)

$$\sigma_{22}(x) = \text{Re}[Z] + x_2 \text{Im}[Z']$$
 (2b)

$$\sigma_{12}(x) = -x_2 \text{Re}[z']$$
 (2c)

$$2\mu u_1(x) = \frac{(\kappa-1)}{(2)} Re[Z^*] - x_2 Im[Z]$$
 (2d)

$$2\mu u_2(x) = \frac{(\kappa+1)}{(2)} Im[Z^*] - x_2 Re[Z]$$
 (2e)

where Z is holomorphic in the plane cut along the real axis. Z' denotes the derivative of Z and Z* the anti-derivative of Z. The material parameter κ has the value of 3-4 ν for problems of plane strain and $(3-\nu)/(1+\nu)$ for plane stress.

The solution for the crack opening point forces, F, acting at the point $x_1 = \xi$ as shown in Fig. 2 is based upon the stress function:

$$Z(z) = -\frac{F}{\pi} \sqrt{\frac{\xi}{z}} \frac{1}{z - \xi}$$
 (3)

with the plane cut along the positive real axis. The branch of $\sqrt{z} = \sqrt{r}e^{i\theta/2}$ is taken such that $\theta \neq 0$ on the upper crack surface and $\theta \neq 2\pi$ on the lower. In the polar coordinates, Z, Z' and Z* have the representations:

$$Z = -\frac{F}{\pi} \sqrt{\frac{\xi}{r}} \frac{1}{\rho} \left[\sin(\theta/2 + \alpha) + i \cos(\theta/2 + \alpha) \right]$$

$$Z' = \frac{F}{\pi} \sqrt{\frac{\xi}{r}} \left\{ \frac{1}{2r\rho} \left[\sin(\theta/2 + \alpha) + i \cos(\theta/2 + \alpha) \right] + \frac{1}{\rho^2} \left[\sin(\theta/2 + \alpha) + i \cos(\theta/2 + \alpha) \right] \right\}$$

$$Z \neq \frac{F}{\pi} \left[(\theta_1 - \theta_2) - i \ln(\rho_1/\rho_2) \right]$$

$$(4e)$$

where $z-\xi = \rho e^{i\alpha}$ as shown in Fig. 2 and

$$\rho_1^2 = r + \xi - 2\sqrt{r\xi} \cos\theta/2 \tag{5a}$$

$$\rho_2^2 = r + \xi + 2\sqrt{r\xi} \cos \theta/2 \tag{5b}$$

$$\theta_1 = \cos^{-1}\left[\frac{\sqrt{r} \cos\theta/2 - \sqrt{\xi}}{\rho_1}\right]$$
 (5c)

$$\theta_2 = \cos^{-1}\left[\frac{\sqrt{r} \cos\theta/2 + \sqrt{\xi}}{\rho_2}\right] \tag{5d}$$

The stresses and displacements may then be obtained by substituting Eq. (4) into Eq. (2). By superimposing the results of a distribution of such point forces through the process of integration, one can construct the solution for the continuous normal stress distribution $\alpha(x_1)$ of Eq. (1) in the forms

$$\sigma_{ij}(x) = \int_{K_{ij}}^{a} (\xi; x) \sigma(\xi) d\xi$$
 (6a)

$$u_{i}(x) = \int_{0}^{a} K_{i}(\xi; x) \sigma(\xi) d\xi$$
 (6b)

where the kernels are given by

$$K_{11} = \frac{1}{\pi \rho} \sqrt{\frac{\xi}{r}} \left[\sin(\theta/2 + \alpha) + \frac{x_2}{2r} \cos(3\theta/2 + \alpha) + \frac{x_2}{\rho} \cos(\theta/2\alpha) \right]$$
 (7a)

$$K_{22} = \frac{1}{\pi \rho} \sqrt{\frac{x}{r}} \left[\sin(\theta/2 + \alpha) - \frac{x_2}{2r} \cos(3\theta/2 + \alpha) - \frac{x_2}{\rho} \cos(\theta/2 + 2\alpha) \right]$$
 (7b)

$$K_{12} = \frac{1}{\pi \rho} \oint_{\mathbf{r}} \left[\sin(\theta/2 + \alpha) \right] \tag{7c}$$

$$K_1 = \frac{1}{2\pi\mu} \left[-\frac{\kappa-1}{2} (\theta_1 - \theta_2) - \frac{\kappa_2}{\rho} \sqrt{\frac{\xi}{r}} \cos(\theta/2 + \alpha) \right]$$
 (7d)

$$R_2 = \frac{1}{2\pi\mu} \left[\frac{\kappa+1}{2} \ln(\rho_1/\rho_2) - \frac{\kappa_2}{\rho} \sqrt{\frac{\xi}{r}} \sin(\Theta/2 + \alpha) \right]$$
 (7e)

The stress intensity factor at the crack tip is defined in the usual manner, i.e.

$$K_{I} = \frac{\lim_{r \to 0} \sqrt{2\pi r} \sigma_{22}(r, \theta)}{e^{-\pi}}$$
(8)

and which, with the use of Eqs. (6a) and (7b), takes the form

$$K_{1} = -\sqrt{\frac{2}{\pi}} \int_{0}^{a} \frac{\sigma(\xi)}{\sqrt{\xi}} d\xi$$
 (9)

2. Equable Problem

The equable problem corresponds to a problem of an unflawed body with the same outer boundary as the original problem and may be formulated by the BIE method [3]; the basics of which are outlined here for subsequent usage.

For any two-dimensional problem on a region bounded by the curve C, the traction and displacement components on C are not independent; rather, for an equilibrated stress state, are related by the contour integral

$$\frac{1}{2}u_{i}(P) = \int_{C} [u_{j}(Q)T_{ij}(P;Q) - \frac{1}{\mu}t_{j}(Q)U_{ij}(P;Q)]dS$$
 (10)

where P and Q denote points on C and dS is the differential arc length of C at Q. The tensors U_{ij} and T_{ij} are singular when P = Q and are known functions associated with the displacement and stress fields due to a concentrated force at P, respectively. They may be written in the forms

$$U_{ij} = \frac{\kappa}{2\pi(\kappa+1)} \left[\delta_{ij} \ln r - \frac{1}{\kappa} r_{i} r_{j} \right]$$
 (11a)

$$T_{ij} = \frac{1}{2\pi(\kappa+1)r} \left\{ [(\kappa-1)\delta_{ij}^{\dagger} + 4r, r,_j] \frac{\partial r}{\partial n} - (\kappa-1)(r,_i^n,_j^{-r},_j^n,_i) \right\}$$
(11b)

The indicated derivatives are with respect to the coordinates at Q, $n_{\dot{1}}$ are the direction cosines of the outward normal to C at Q and now, r is the distance between P and Q.

In principle, Eq. (10) is sufficient to determine, to within a rigid body displacement, the unspecified portion of the traction and/or displacement components on the boundary of any well posed problem. For any but the most simple problems, closed form solutions are unavailable, but accurate numerical approximations are possible. Algorithms for the approximate evaluation of Eq. (10) have been developed [3,4] and successfully applied to many problems.

Assuming, for explanatory purposes, that a solution of Eq. (10) has been achieved, then a full complement of data on the boundary would be known. This complete set of boundary data may then be used to determine the elastic field variables at any interior point of the body according to the relations

$$u_{i}(x) = \int_{C} [u_{j}(Q)T_{ij}(x;Q) - \frac{1}{\mu} t_{j}(Q)U_{ij}(x;Q)]dS$$
 (12a)

$$\frac{1}{\mu} \sigma_{ij}(x) = \int [u_k(Q)S_{kij}(x;Q) - \frac{1}{\mu} t_k(Q)D_{kij}(x;Q)]dS$$
 (12b)

where the tensors D_{kij} and S_{kij} are related to U_{ij} and T_{ij} via the stress-displacement equations of linear elasticity. Explicit forms of D_{kij} and S_{kij} are given in Reference 1.

3. Simultaneous Solution of the Perturbed and Equable Problems

The equations governing the perturbed and equable problems can then be combined to give a set of coupled integral equations solvable for certain unknowns from each problem in terms of the specified data of the original problem. Accordingly, if the i-th traction component t_i is specified on that portion of C denoted by C_i^t and similarly the i-th displacement component on C_i^u , then, if the sum of the perturbed and equable problems is to produce the original problem, one must require that

$$u_{i}^{P}(P)+u_{i}^{E}(P) = u_{i}(P), P \in C_{i}^{u}$$
 (13a)

$$t_{i}^{P}(P)+t_{i}^{E}(P) - t_{i}(P) , P \in C_{i}^{t}$$
 (13b)

$$\sigma(\mathbf{x}_1) + \sigma_{22}^{E}(\mathbf{x}_1, 0) - \sigma_0(\mathbf{x}_1), 0 \le \mathbf{x}_1 \le \mathbf{a}$$
 (13c)

The superscripts P and E denote variables associated with the perturbed and equable problems, respectively, and $\sigma_0(x_1)$ is the prescribed stress on the crack in the original problem.

Now, substitution of Eqs. (13a) and (13b) into Eq. (10) leads to a boundary integral equation involving variables from both the perturbed and equable problems in terms of prescribed data in the original problem. With the notation

$$\mathbf{u_i^k(P)} = \begin{cases} \mathbf{u_i^k} & (P), \ P \in C_i^t \\ -\mathbf{u_i^k} & (P), \ P \in C_i^u \end{cases}$$
(14a)

$$\mathbf{t_{i}^{*}(P)} = \begin{cases} -\mathbf{t_{i}^{P}(P)}, \ P \in C_{i}^{t} \\ \mathbf{t_{i}^{E}(P)}, \ P_{c}C_{i}^{u} \end{cases}$$
(14b)

The resulting equation is readily put into the form

$$\int_{\mathbf{C}} \left[u_{\mathbf{j}}^{\star}(Q) T_{\mathbf{i}\mathbf{j}}(P;Q) - \frac{1}{\mu} t_{\mathbf{j}}^{\star}(Q) U_{\mathbf{i}\mathbf{j}}(P;Q) \right] dS - \frac{1}{2} u_{\mathbf{i}}^{\star}(P) - \int_{\mathbf{C}} \left[\frac{1}{\mu} t_{\mathbf{j}}(Q) U_{\mathbf{i}\mathbf{j}}(P;Q) - \frac{1}{2} t_{\mathbf{j}}^{\dagger}(P) \right] - \frac{1}{2} t_{\mathbf{j}}^{\dagger}(P) \right] - \frac{1}{2} \left[0 - u_{\mathbf{i}}^{\dagger}(P) \right]$$
(15)

The bracketed terms in Eq. (15) are to be interpreted as follows. The upper variables are to be used when the k-th component of the boundary variable is contained in \mathbf{C}_k^t and the lower when contained in \mathbf{C}_k^u .

In a similar fashion, the evaluation of the i=j=2 component of Eq. (12b) at $x=(\xi,0),\ 0\le \xi\le a$, while using Eqs. (13a) and (13b) produces

$$\frac{1}{\mu}\sigma_{22}^{E}(\xi,0) - \int_{C} [u_{k}^{E}(q)s_{k22}(\xi,0;q) - \frac{1}{\mu}\epsilon_{k}^{E}(q)D_{k22}(\xi,0;q)] ds = 0$$
 (16)

With Eqs. (13c) and (14), this relation is put into the form

$$\int_{C} \left[u_{k}^{\star}(Q) S_{k22}^{-}(\xi,0;Q) - \frac{1}{\mu} \epsilon_{k}^{\star}(Q) D_{k22}^{-}(\xi,0;Q) \right] dS + \frac{1}{\mu} \sigma(\xi) - \int_{C} \left[\frac{1}{\mu} \epsilon_{k}^{-}(Q) D_{k22}^{-}(\xi,0;Q) \right] dS + \sigma_{0}(\xi)$$
(17)

Then Eqs. (6) are evaluated at x = P to obtain, with the definitions in Eqs. (14):

Equations (15), (17) and (18) are a set of integral equations, coupled through the functions u_i^* and t_i^* on C and σ on the line of the crack, representing the solution to the original problem in terms of the prescribed boundary values u_i , t_i and σ_0 . If a solution to these equations were achieved, σ would be known and then Eq. (9) could be used to evaluate the stress intensity factor.

SECTION III

NUMERICAL SOLUTION

1. Reduction to Linear Algebraic Equations

An approximate solution to the governing equations (15), (17), and (18) may be obtained as follows. Let the outer boundary C be approximated by M straight line segments $C_m(m=1,2,\ldots,M)$ with a segment division at the intersection of the line of the crack and C, i.e. at the point (a,0). Assume that the tractions and displacements t_i^* , u_i^* , t_i and u_i have the constant values t_i^* , u_i^* , t_i^* and u_i^* respectively on C_m and let the boundary point P occupy the M segment midpoints denoted by $P_1(1=1,2,\ldots,M)$ as shown in Fig. 3. Under these assumptions, integrals over the outer boundary, such as the first term of Eq. (15), may be represented by relations of the type

$$\int_{C} u_{j}^{\star}(Q) T_{ij}(P,Q) dS \approx \sum_{m=1}^{M} \int_{C_{m}} u_{j}^{\star}(Q) T_{ij}(P,Q) dS = \sum_{m=1}^{M} u_{j}^{\star m} \int_{C_{m}} T_{ij}(P,Q) dS$$
(19)

The unknown boundary variable has thus been removed from the integral and the remaining kernel is a known function of the linearized boundary geometry, so that for a given point P and segment C_{m} , the integration can be performed in closed form or, if necessary, approximated. Note

that once in every complete integration over C, P will be on the integration path C_m , i.e. $P = P_m \varepsilon C_m$, in which case T_{ij} is singular and must be evaluated in the sense of a Cauchy principal value. In the above fashion, all integrals over the outer boundary in Eqs. (15) and (17) may be reduced to algebraic products with coefficients determinable from the linearized boundary geometry.

The integrals on the line of the crack in Eqs. (18) are to be evaluated by Gaussian quadrature. Likewise, the evaluation of the stress intensity by Eq. (9) is to be approximated by a Gaussian sum. The singularity in the integral of Eq. (9) is removed by the transformation $\xi + \xi^2$ so that the stress intensity factor is given by

$$\kappa_{I} = -2\sqrt{\frac{2}{\pi}} \int_{0}^{\sqrt{a}} \sigma(\xi^{2}) d\xi$$
 (20)

Hence, if ξ_{κ} and w_{κ} (k=1,2,..., N) denote the Gaussian modes and weights of order N, Eq. (20) is approximated by

$$K_{I} = -\sqrt{\frac{2a}{\pi}} \sum_{k=1}^{N} \sigma[a(1+\xi_{k})^{2}/4] w_{k}$$
 (21)

To evaluate Eq. (21) it is necessary to know the N values of σ at $\xi_k = a(1+\xi_k)^2/4$. Accordingly, the same transformation, $\xi+\xi^2$, is applied to the integration parameter of Eqs. (18) so that, for example, Eq. (18a) will have the approximate representation

$$t_{i}^{\star}(P) = \sqrt{\frac{a}{2}} \sum_{k=1}^{N} K_{ij}(\zeta_{k}; P) n_{j}(P) \sigma(\zeta_{k}) \zeta_{k} w_{k}$$
(22)

Note that under the assumption that P occupy the discrete locations $P_m(m=1,2,\ldots,M)$ corresponding to the midpoints of the M outer boundary

segments and with boundary segments beginning and ending at the crack mouth, the possibility of the kernel $K_{ij}(\xi;P)$ being singular is precluded.

Consistent with the above assumptions and resulting approximations, as exemplified by Eqs. (19) and (22), the set of coupled integral equations (15), (17) and (18) may be reduced to a set of 4M+N linear, algebraic equations whose solution is sufficient to determine the 4M discrete values of the boundary variables t_i^{*m} and u_i^{*m} (i=1,2; m=1,2, ... M) and the N values of $\sigma(\xi_k)$ (k = 1,2, ..., N). Upon evaluation of Eqs. (15) and (18) at P_m (m = 1,2, ..., M) and Eq. (17) at the nodes $\zeta_k = a(1+\xi_k)^2/4$, these equations have the approximate representation

$$\sum_{m=1}^{M} \left[u_{j}^{*m} \Delta T_{ij}^{lm} - \frac{1}{\mu} c_{j}^{*m} \Delta U_{ij}^{lm} \right] = \sum_{m=1}^{M} \begin{bmatrix} \frac{1}{\mu} c_{j}^{m} \Delta U_{ij}^{lm} \\ -u_{j}^{m} \Delta T_{ij}^{lm} \end{bmatrix}$$
(23a)

$$\sum_{m=1}^{\mathbf{M}} \left[\mathbf{u}_{\mathbf{i}}^{*m} \Delta \mathbf{S}_{\mathbf{i}}^{km} - \frac{1}{\mu} \mathbf{t}_{\mathbf{i}}^{*m} \Delta \mathbf{D}_{\mathbf{i}}^{km} \right] + \frac{1}{\mu} \sigma \left(\zeta_{\mathbf{k}} \right) = \sum_{m=1}^{\mathbf{M}} \left[\frac{1}{\mu} \mathbf{t}_{\mathbf{i}}^{m} \Delta \mathbf{D}_{\mathbf{i}}^{km} \right] + \frac{1}{\mu} \sigma_{0} (\zeta_{\mathbf{k}})$$

$$(23b)$$

$$t_{i}^{*1} + \sqrt{a} \sum_{k=1}^{N} K_{ij} (\zeta_{k}; P_{1}) n_{j} (P_{1}) \sigma(\zeta_{k}) \sqrt{\zeta_{k}} w_{k} = 0, P_{1} \varepsilon C_{i}^{t}$$
(23c)

$$\mathbf{u}_{i}^{*1} + \sqrt{a} \sum_{k=1}^{N} \mathbf{K}_{i}(\zeta_{k}; \mathbf{P}_{1}) \sigma(\zeta_{k}) \sqrt{\zeta_{k}} \mathbf{w}_{k} = 0 \qquad \mathbf{P}_{1} \in \mathbf{C}_{i}^{\mathbf{u}}$$
 (23d)

where the coefficients are defined by

$$\Delta U_{ij}^{lm} = \int U_{ij}(P_1;Q) dS$$
 (24a)

$$\Delta T_{ij}^{lm} = \int_{C_{m}}^{m} T_{ij}(P_{1};Q)dS - \frac{1}{2}\delta_{ij}\delta_{lm}$$
(24b)

$$\Delta D_{i}^{km} = \int D_{i22}(\zeta_{k}, 0; Q) dS \qquad (24c)$$

$$\Delta D_{i}^{km} = \int_{0}^{\infty} D_{i22}(\zeta_{k}, 0; Q) dS$$

$$\Delta S_{i}^{km} = \int_{0}^{\infty} S_{i22}(\zeta_{k}, 0; Q) dS$$

$$C_{m}$$
(24c)
(24d)

When a solution of Eqs. (23) have been achieved, $\sigma(\xi_k)$ will be known and the stress intensity factor can be evaluated by Eq. (21).

A FORTRAN computer program was written (Appendix) to evaluate the coefficients defined by Eqs. (24), assemble and solve Eqs. (23), for outer boundary regions of arbitrary shape, including regions of multiple connectivity. The algorithms for the evaluation of Eqs. (24a) and (24b) were developed in Reference (5). The integrals of Eqs. (24c) and (24d) were approximated by Simpsons rule. A Gaussian elimination procedure was employed for the simultaneous solution of the algebraic equations.

Outer Boundary Modeling

The accuracy of the solution provided by Eqs. (23) for the integral equations they approximate is obviously dependent upon the discretization of the outer boundary. In general, when the BIE method is implemented under the assumptions of linear boundary segments and constant boundary variables on the segments, the following two modeling considerations are relevant.

- (a) The "best" results are obtained when the ratio of the lengths of adjacent boundary segments is within the range of 0.5 to 2.0.
- (b) The resolution of the stresses at an interior point of a body by Eq. (12b) deteriorates significantly when the interior point is within approximately one segment's length of the boundary segment itself.

Consideration (b) has an important bearing on the modeling of the outer boundary of the present formulations since the stress component σ_{22} must be evaluated by Eq. (23b) at points on the crack in the vicinity of the outer boundary.

The point nearest to the outer boundary at which Eq. (23b) is to be evaluated is most likely, but not necessarily, the last mode on the crack $\zeta_N = a(1+\xi_N)^2/4$. The nearby boundary segments, i.e. those beginning and ending at the crack mouth, should thus be at least $a[1+(1+\xi_N)^2/4]$ units in length. To avoid the usage of very small segments at the crack mouth, it is thus desirable to keep N to a minimum, thereby increasing the distance from the last node on the crack to the outer boundary. However, retention of a sufficient number of terms in the Gaussian sums of Eqs. (23c) and (23d) is required to give an accurate evaluation of these equations. A compromise is required.

Assuming seven to ten terms in the Gaussian quadratures will provide adequate resolution of the integrals on the crack, the length of the boundary segments at the crack mouth should correspondingly be about 0.050a to 0.026a units in length for N = 7 and N = 10, respectively. This and consideration (a) were followed in modeling the subsequent illustrative example problems.

SECTION IV

ILLUSTRATIVE EXAMPLES

Infinite Plate - Pressurized Crack

An analytical solution may be obtained for the problem of a semiinfinite crack in an infinite plate with a uniform pressure $P_{\hat{Q}}$ over a portion of the crack near the tip as shown in Fig. 4. The analytical solution to this problem is subsequently used to simulate a finite plate problem from which to gauge the accuracy of the present formulation and numerical solution.

The solution to the problem of Fig. 4 may be obtained by the integration of Eqs. (6) with $\sigma(\xi) = -P_0$ or, as is done here, from Eqs. (2) using the stress potential function

$$Z(z) = \frac{2P_0}{\pi} \left[i \sqrt{\frac{a}{z}} - \tan^{-1} i \sqrt{\frac{a}{z}} \right]$$
 (25)

whose derivative and anti-derivative are

$$Z'(z) = \frac{P_0}{\pi} \left(\frac{a}{z}\right)^{3/2} \frac{i}{z-a}$$
 (26a)

$$Z^{*}(z) = \frac{2P_{0}a}{\pi} \left[i\sqrt{\frac{z}{a}} + (1 - \frac{z}{a}) + \tan^{-1} i\sqrt{\frac{z}{a}} \right]$$
 (26b)

substituting Eqs. (25) and (26) into Eqs. (2), one obtains

$$\sigma_{11} = \frac{P_0}{\pi} \left[2 \sqrt{\frac{a}{r}} \sin \theta / 2 + (\theta_1 - \theta_2) - \left(\frac{a}{r} \right)^{3/2} \frac{x_2}{\rho} \cos (3\theta / 2 + \alpha) \right]$$
 (27a)

$$\sigma_{22} = \frac{P_0}{\pi} \left[2\sqrt{\frac{a}{r}} \sin \theta/2 + (\theta_1 - \theta_2) + \left(\frac{a}{r}\right)^{3/2} \frac{x_2}{\rho} \cos (3\theta/2 + \alpha) \right]$$
 (27b)

$$\sigma_{12} = \frac{P_0}{\pi} \left(\frac{\mathbf{a}}{\mathbf{r}} \right)^{3/2} \frac{\mathbf{x}_2}{\rho} \sin(3\theta/2 + \alpha) \tag{27c}$$

$$2\mu u_1 = \frac{P_0}{\pi} \left\{ \frac{\kappa - 1}{2} \left[\rho(\theta_1 - \theta_2) \cos \alpha + \rho \sin \alpha \ln \frac{\rho_1}{\rho_2} - 2\sqrt{ar} \sin \theta/2 \right] \right\}$$

$$-2y\sqrt{\frac{a}{r}}\cos \theta/2 + \frac{x_2}{\rho} \ln \frac{\rho_1}{\rho_2}$$
 (27d)

$$2\mu u_{2} = \frac{P_{0}}{\pi} \left\{ \frac{\kappa+1}{2} \left[\rho(\theta_{1} - \theta_{2}) \sin \alpha - \rho \cos \alpha \ln \frac{\rho_{1}}{\rho_{2}} + 2\sqrt{ar} \cos \theta/2 \right] - 2^{x_{2}} \sqrt{\frac{a}{r}} \sin \theta/2 - x_{2}(\theta_{1} - \theta_{2}) \right\}$$
(27e)

where again $z = re^{i\theta}$, but now, $z-a = \rho e^{i\alpha}$ as shown in Fig. 4 and

$$\rho_1^2 = r + a + 2\sqrt{ar} \cos \theta/2 \tag{28a}$$

$$\rho_2^2 = r + a - 2\sqrt{ar} \cos \theta/2 \tag{28b}$$

$$\theta_1 = \cos^{-1} \left[\frac{\sqrt{r} \cos \theta/2 + \sqrt{a}}{\rho_1} \right]$$
 (28c)

$$\theta_2 = \cos^{-1} \left[\frac{\sqrt{r} \cos \theta/2 - \sqrt{a}}{\rho_2} \right]$$
 (28d)

The stress intensity factor is determined by Eq. (9) and has the value

$$K_{I} = -\sqrt{\frac{2}{\pi}} \int_{0}^{a} \frac{P_{0}}{\sqrt{\xi}} d\xi = 2\sqrt{\frac{2a}{\pi}} P_{0}$$
 (29)

A finite region surrounding the crack tip (dotted line in Fig. 4) was modeled by the rectangular boundary model shown in Fig. 5. Using, as input boundary conditions, the traction components at the boundary segment **midpoints** as determined by Eqs. (27a), (27b), and (27c), the numerical solution was performed with the parametric values $P_0 = \mu$, a = 0.5 and N = 7. By the present formulation, the exact solution to this problem is provided by a perturbed problem with the same tractions

at the outer boundary as the input values, a pressure $P_0 = \mu$ on the crack and a null equable problem, i.e. the solution is totally accounted for by the perturbed problem. The numerical solution, along with the input traction components, are shown in Table 1 and show good agreement between the input values and the calculated perturbed problem solution. The most deviation occurs at the segments near the crack mouth, as may be expected in view of the assumption of constant variation in the boundary variables over each segment, since the stress components in the perturbed problem vary rapidly in the vicinity of a loaded crack surface, especially at a point of discontinuity of the load.

Similarly, the displacements determined by Eqs. (27d) and (27e) and a unit pressure on the crack were used as input and the numerical solution performed. The input and calculated solution are presented in Table 2. Again there is good agreement between the input displacement components and those determined for the perturbed problem with similar deviation from zero of the calculated tractions near the crack mouth.

The crack surface stresses and stress intensity factors for both the traction and displacement problems are shown in Table 3. The stresses are seen to be accurately determined with again the maximum error near the crack mouth. For the traction problem, the stress intensity factor is in error by only 1.01% while for the displacement problem, agreement to four decimal places is observed.

2. Center Cracked Square Plate - Various Boundary Conditions

Again using the rectangular boundary model of Fig. 5, the problem of a square plate with a center crack under the three loading conditions

(i) uniform tension (Fig. 6), (ii) uniform displacement with no shear, and (iii) uniform displacement with clamped ends was solved with the edge crack program. As shown for case (i) in Fig. (6), one half the center cracked plate was simulated with the edge crack model by application of the above three boundary conditions on the upper and lower boundaries with zero x_1 -direction displacements and x_2 -direction tractions on the vertical centerline. The numerical solution was performed for a half-crack length a=0.5 and with $\sigma_{22}/\mu=1$ for case (i) and unit normal displacements and $\nu=0.3$ for cases (ii) and (iii). The resulting stress intensity factors are shown in Table 4, along with published [6] values for comparison.

SECTION V

CONCLUSIONS

The coupled integral equation formulation of a crack problem in an infinite medium with a problem in an unflawed medium has been shown to be an effective method to solve edge crack problems. This type of formulation allows for a direct and accurate evaluation of stress intensity factors and may be applied to problems of arbitrary shape. Although only the mode I problem was addressed here, the method may be easily extended to include the mode II and combined mode problems by incorporation of the mode II perturbed problem equations into the formulation.

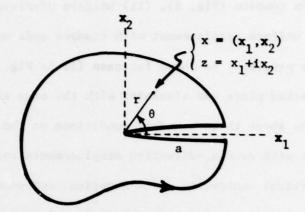


Figure 1. Arbitrary Body with Edge Crack and Coordinate System.

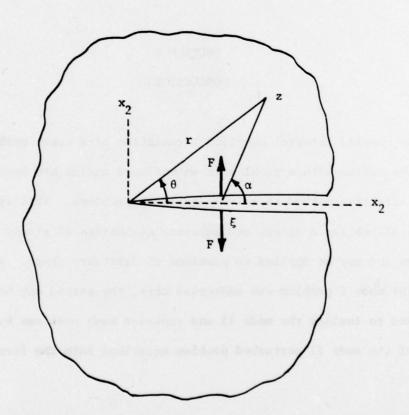


Figure 2. Point Force on Semi-Infinite Crack in an Infinite Medium.

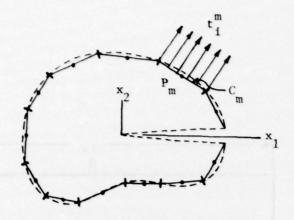


Figure 3. Outer Boundary Modeling Scheme.

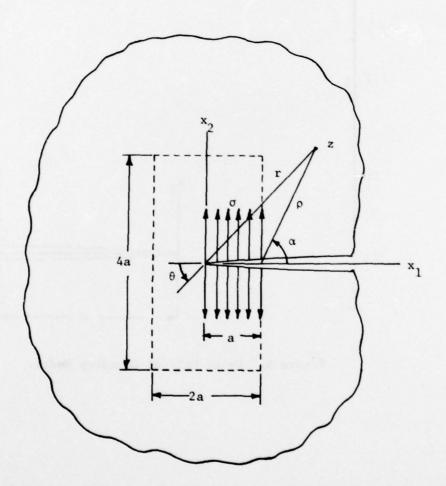


Figure 4. Infinite Plane with Uniform Pressure on the Crack.

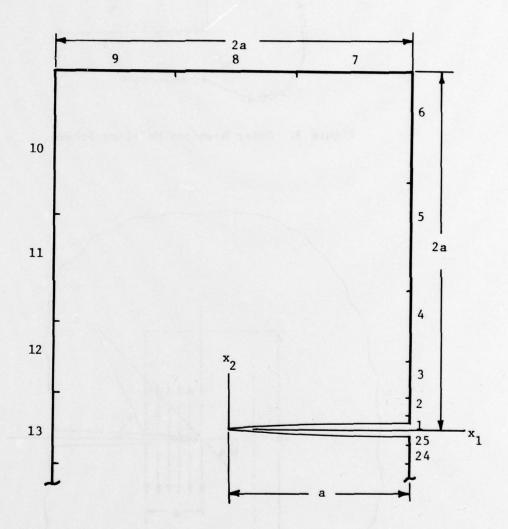


Figure 5. Upper Half of Boundary Model.

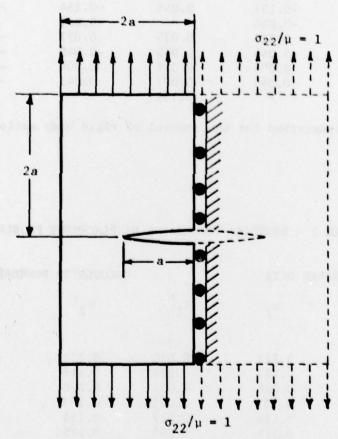


Figure 6. Center Cracked Plate Under Uniform Tension Modeled with Edge Crack Problem.

TABLE 1 - BOUNDARY VALUES FOR TRACTION PROBLEM

SEGMENT	PRESCRI	BED DATA		CALCULATED BO	UNDARY VALUES	
NUMBER	t ₁	^t 2	t ₁ ^P	t ₂ P	u ₁ E	u ₂ E
1	-0.476	-0.318	-0.640	-0.285	0.006	0.001
2	-0.405	-0.312	-0.402	-0.376	0.003	0.003
3	-0.275	-0.284	-0.278	-0.288	0.001	0.002
4	-0.090	-0.190	-0.090	-0.193	-0.001	0.003
5	0.020	-0.064	0.021	-0.064	-0.004	0.003
6	0.034	-0.003	0.035	-0.002	-0.007	0.004
7	0.046	-0.151	0.048	-0.154	-0.009	0.003
8	*	-0.066		-0.067	*	0.000
9	0.074	0.026	0.075	0.027	-0.008	-0.002
10	0.044	-0.045	0.045	-0.045	-0.007	-0.003
11	0.021	0.004	0.021	0.004	-0.003	-0.003
12	-0.075	0.040	-0.077	0.041	-0.001	-0.002
13	-0.137	*	-0.139	*	0.000	*

^{*} Displacements prescribed for the removal of rigid body motion

TABLE 2 - BOUNDARY VALUES FOR DISPLACEMENT PROBLEM

SEGMENT	PRESCRI	BED DATA		CALCULATED BOUNDARY VALUES						
NUMBER	^u 1	^u ₂	u ₁ P	^u 2 ^P	t ₁ ^E	t ₂ ^E				
1	0.009	0.222	0.010	0.221	-0.013	0.128				
2 3	0.022	0.217	0.022	0.218	0.003	-0.033				
	0.031	0.208	0.031	0.208	0.001	0.006				
4 5	0.028	0.187	0.028	0.187	0.000	0.000				
5	0.017	0.154	0.017	0.154		1				
6	0.008	0.125	0.008	0.125						
7	0.000	0.108	0.000	0.108		1				
8	-0.008	0.083	-0.008	0.083						
9	-0.007	0.053	-0.007	0.053						
10	0.000	0.035	0.000	0.035						
11	0.014	0.019	0.014	0.019						
12	0.030	0.007	0.030	0.007						
13	0.036	0.000	0.036	0.000						

TABLE 3 - STRESS ON CRACK SURFACES IN PERTURBED PROBLEM

AND STRESS INTENSITY FACTORS

NODE	GAUSSIAN NODE [§] n	CRACK COORDINATE $\zeta_n = 0.5(1+\xi_n)^2/4$	STRESS	ON CRACK TRACTION PROBLEM	SURFACE DISPLACEMENT PROBLEM
1	-0.9491	0.0003	-1.0	-0.9995	-1.0001
2	-0.7415	0.0084	-1.0	-0.9997	-1.0001
3	-0.4058	0.0441	-1.0	-1.0007	-1.0001
4	0.0000	0.1250	-1.0	-1.0030	-1.0001
5	0.4058	0.2471	-1.0	-1.0071	-1.0003
6	0.7415	0.3791	-1.0	-1.0162	-1.0011
7	0.9491	0.4749	-1.0	-1.0900	-0.9961
STRESS IN	TENSITY FACTO	OR .	1.1284	1.1398	1.1284

TABLE 4 - STRESS INTENSITY FACTORS FOR A SQUARE
PLATE UNDER VARIOUS BOUNDARY CONDITIONS

	BlE	PUBLISHED [6]	PERCENT DIFFERENCE
TENSION	1.605	1.672	4.01
DISP. (NO SHEAR)	2.844	2.883	1.35
DISP. (CLAMPED)	2.925	3.067	4.63

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- 2. H.M. Westergaard, "Bearing Pressures and Cracks", Trans. ASME, Vol. 61, 1939, pp. A49-A53.
- F.J. Rizzo, "An Integral Equation Approach to Boundary Value Problems of Classical Elastostatics", Quart. Applied Math., Vol. 25, No. 1, April 1967, pp. 83-95.
- 4. P.C. Riccardella, "An Improved Implementation of the Boundary-Integral Technique for Two-Dimensional Elasticity Problems", Carnegie-Mellon University, Report SM-72-26, September 1972.
- T.J. Rudolphi, "An Integral Equation Solution for a Bounded, Plane, Elastic Body Containing a Crack: In-Plane Deformations", Ph.D. Thesis, University of Illinois, 1977.
- M. Isida, "Effect of Width and Length on Stress Intensity Factors of Internally Cracked Plates Under Various Boundary Conditions", International Journal of Fracture Mechanics, Vol. 7, No. 3, September 1971, pp. 301-316.

APPENDIX

```
PROGRAM TOEMEC (INPUT. OUTPUT. TAPE5=INPUT. TAPE6=OUTPUT)
000000000
                     THE ARRAYS A(I) AND I(I) MUST BE DIMENSTONED AS
                     A (24M**2*10MN+N**2+18M+5N) . I (64+N+N3)
                     WHERE:
                                M = NO UF OUTER BNDY SEGMENTS
                                N = NO OF GAUSSIAN INTEGRATION POINTS ON CRACK
                                NA = NO OF BOUNDARIES
C
      DIMENSION A(17284) . I (158)
      DIMENSION LABEL (20)
C
CC
                     INPUT AND SUTPUT PROBLEM LABEL
      READ (5.1000) LABEL
      WRITE(6,1001) LABEL
C
C
                    FN.N.M TUPNI
C
      READ(5.1002) M.N.N3
      M2=2+M
      NOE 7=4+4+N
      I1=1
      DECK+DECK+DEC
      SH#FBCN+SI=EI
      14=13+NDEQ
      15=14+M2
      I6=I5+H+3
      17=16+M+3
      18=17+N
      19=18+N
      I10=19+4
                                    THE PROPERTY PRACTICALLY
                                   FROM CUIT FURNISHED TO DDC
      I11=I10+M
      I12=I11+NOEQ
      113=1
      I14=I13+M2
      115=114+NOEQ
      CALL PRG4(M+N+H2+NB+NOEQ+A(I1)+A(I2)+A(I3)+A(I4)+A(I5)+A(I5)+A(I6)+A(I7)
     1.A(18).4(19).A(110).A(111).A(112).I(113).I(114).I(115))
C
1000
      FORMAT (20A4)
1001
      FORMAT (1H1 .// . T35 . 20 A4 .//)
1002
      FORMAT (315)
                                           THIS PAGE IS BEST QUALITY PRACTICARLE
      STOP
                                           FROM COPY FURNISHED TO DDC .
      END
```

```
SUBROUTINE PROMIMONOMEON SONDEQONOMO POR SONDER SON
              1 ADTYPE . II . LNB)
                REAL KAPPA . N1 . N2 . K1 . K2 . K11 . K12 . K22
                DIMENSION A(NUEQ.NOEQ).B(NOEQ.M2).RHS(NOEQ).T(M2).X(4.3).Y(M.3).
              1XC(N), W(N), PHI (M), DELS(1), CC (NOEQ), SC (N)
              2,0(3),THETA(3),SN(3),CS(3)
                INTEGER BOTYPE (M.2) . II (NOEQ) . LNB (NB)
                PI=3.141592653590
                M3=4+3
                M4=M+4
C
                                                   INPUT CRACK LENGTH AND MATERIAL PROPERTY
C
                READ (5,1000) AA, KAPPA
                WRITE(6, 1001) M.N.NB.AA.KAPPA
                C=1.0/(2.0*PI*(KAPPA+1.0))
C
C
                                                  INPUT BOUNDARY COORDINATES AND GAUSSIAN NODES
C
                CALL BOUND (M.MZ.NR.X.Y.LNB)
                READ (5.1002) (XC(I).W(I).I=1.N)
C
C
                                                  CALCULATE MID AND END SEGMENT COORDINATES
                NSEG=0
                NBX=1
                00 130 I=1.M
                IF (I.NE.LNB(NBX)) 30 TO 110
                X(I.3) = X(I-NSEG.1)
                Y(I.3) = Y(I-NSEG.1)
                N9X=N9X+1
                                                                                   THIS PAGE IS BEST QUALITY PRACTICABLE
                NSEG=0
                                                                                   FROM COPY FURNISHED TO DDC
                GO TO 120
                NSEG=NSEG+1
110
                X(I,3) = X(I+1,1)
                Y(I,3) = Y(I+1,1)
120
                X(I,2) = (X(I,1) + X(I,3))/2.0
130
                Y(I,2) = (Y(I,1) + Y(I,3))/2.0
C
C
                                                  CALCULATE DELS(I) AND PHI(I)
C
                DO 140 I=1.4
                DELS(I) = DIST(X(I+1)+Y(I+1)+X(I+3)+Y(I+3))
140
                PHI (I) = ANGL (Y (I . 1) . X (I . 3) . Y (I . 3) . X (I . 1))
C
C
                                                  INPUT BOUNDARY CONDITIONS AND SUTPUT PROBLEM DATA
C
                CALL ADATA (4. M2. KAPPA. AA. X.Y.PHI. BOTYPE.T)
                WRITE(5.1006)
                DO 145 I=1.M
                WRITE(6.1007) I.X(I.1).X(I.2).X(I.3).PHI(I)*180.0/PI.DELS(I).
             130TYPE(I,1),90TYPE(I,2),T(I),7(I+M),Y(I,1),Y(I,2),Y(I,3)
C
                                                  ZERO A(I.J) AND B(I.J)
C
                00 155 I=1. NOEQ
                DO 150 J=1. NOEQ
```

```
150
      A(I.J)=0.0
      DO 155 J=1.42
155
      3(I.J) =0.0
C
                    CALCULATE COEFFICIENTS OF RIE
C
C
      00 180 I=1.4
      00 180 J=1.M
      DO 160 K=1.3
      0(K)=015T(X(I.2).Y(I.2).X(J.K).Y(J.K))
      IF (I.EJ.J.AND.K.EJ.2) 30 TO 160
       THETA(K) = ANGL(X(I \cdot 2) \cdot Y(I \cdot 2) \cdot X(J \cdot K) \cdot Y(J \cdot K))
      SN(K)=SIN(2.0+THETA(K))
      CS(K)=COS(2.0+THETA(K))
160
      CONTINUE
      DS=DELS(J)
      DT=THETA(3)-THETA(1)
      A(I.J) = C + ((KAPPA+1.0) + DT + SN(3) - SN(1))
      A(I.J+M) =C*((KAPPA-1.0)*ALOG(D(3)/D(1))+CS(1)-CS(3))
      A(I+M.J) =C+((1.0-KAPPA)+ALOG(D(3)/D(1))+CS(1)-CS(3))
      A(I+M.J+M) =C*((KAPPA+1.0)*OT-SN(3)+SN(1))
      IF (I.E2.J) GO TO 170
      A(I+J+M2)=C+DS+(KAPPA+ALOG(D(1)+D(2)++4+D(3))-(6.0+25(1)+4.0*C5(2)
     1+CS(3))/2.0)/6.0
      A(I.J+M+M2)=A(I+M.J+M2)=-C+OS+(SN(1)+4.0+SN(2)+SN(3))/12.0
      A(I+M+J+M+M2)=C+OS+(KAPPA+ALOG(D(1)+D(2)++4+D(3))-(6.7-2S(1)-4.0*C
     15(2)-05(3))/2.0)/6.0
      IF (ABS(DT).LE.PI) GO TO 180
      DT1=THETA(2)-THETA(1)
      DT2=THETA(3)-THETA(2)
      IF (ABS(OT1).GT.PI) OT=DT-2.0*PI*SIGN1(OT1)
      IF (ABS(DT2).GT.PI) DT=DT-2.0*PI*SIGN1(DT2)
      A(I,J)=2+((KAPPA+1.0;+OT+SN(3)-SN(1))
      A(I+M,J+M)=C+((KAPPA+1.0)+DT-SN(3)+SN(1))
      GO TO 190
170
      TEMP=D(1) * (ALOG(D(1)) -1.0) +D(3) * (ALOG(D(3)) -1.0)
      A(I.I+M2)=C*(KAPPA*TEMP-DS*(2.0+CS(1)+CS(3))/4.0)
      A(I.I+M3) = A(I+M.I+M2) = -C+DS+(SN(1)+SN(3))/4.0
      A(I+M+I+H3)=C*(KAPPA*TE4P-DS*(2.0-CS(1)-CS(3))/4.0)
      A(I.I) = A(I.I) - C*PI* (KAPPA+1.0) *SIGN1(DT)-0.5
      A(I+M.I+M) = A(I+M.I+M) - C+PI+(KAPPA+1.0) +SIGN1(DT) -0.5
      CONTINUE
180
C
                    CALCULATE COEFFICIENTS FOR THE INTERNAL STRESSES
C
      DO 200 I=1.N
      XI=0.25+AA+(1.0+XC(I))++2
      DO 200 J=1.4
                                           THIS PAGE IS BEST QUALITY PRACTICABLE
      N1=COS (PHI (J))
                                          FROM COPY FURNISHED TO DDC
      N2=SIN(PHI(J))
      DS=DELS(J)
      XB=X(J.1)
      YB=Y(J.1)
      XE=X(J,3)
      YE=Y (J.3)
      RB=DIST(XI.0.0.XB.Y3)
      RE=DIST(XI.O.O.XE.YE)
```

```
DL = (ABS((YE-YB) *XI+XE+Y3-X3*YE))/DS
      1 (5** JC-5**87)28 A) TFC2=1X
      X2=SORT(A3S(RE **2-DL **2))
      RMIN=AMIN1 (RB.RE)
      IF (X1.LT.US.AND.X2.LT.JS) RMIN=DL
      NSU3=2*(1+INT(6.0*)S/RMIN))
      R13=(X3-XI)/28
      R1E=(XE-XI)/RE
      E5/E4=E28
      RZE=YE/RE
      51=S122(C.KAPPA.RB.R1B.R2B.N1.N2)+S122(C.KAPPA.RF.R1E.R2E.N1.N2)
      $2=$222(C.KAPPA.R8.R18.R28.N1.N2)+$222(C.KAPPA.RE.F1E.R2E.N1.N2)
      01=0122(C,KAPPA,RB,R18,R28)+0122(C,KAPPA,RE,R1E,R2E)
      D2=D222(C,KAPP4.RB.R1B.R2B)+D222(C.KAPP4.RE.R1E.R2E)
      WT=2.0
      IS=1
      00 190 K=2.NSUR
      12=-12
                                         THIS PAGE IS BEST QUALITY PRACTICABLE
      XX = XP + (XE - XB) + (K-1) / NSUP
                                         FROM COPY FURNISHED TO DDC
      FUZN ( 1-X) + (EY-3Y) + (K-1) / NSUR
      R=DIST(XI.O.C.XX.YY)
      R1=(XX-XI)/R
      RZ=YY/R
      WT=WT-2.0*I3
      S1=S1+NT+S122(C.KAPPA.R.R1.R2.N1.N2)
      S2=S2+WT+S222(C.KAPPA.R.R1.R2.N1.N2)
      01=01+WT +0122(C.KAPPA.R. R1. R2)
190
      02=02+NT+0222(C.KAPPA.R.21.22)
      ALI+M4.J
                )=DS*S1/(3.0*NSUR)
      (EU2N*0.6)/52*20=( N+L+++1)A
      4(I+M4.J+M2)=DS+D1/(3.0*NSUB)
005
      4(I+M4.J+M3)=DS*D2/(3.0*NSUR)
3
                    CALCULATE KERNELS FOR PERTURBED PROBLEM
G
C
      DO 250 I=1.M
      XX=X(I.2)
      YY=Y (1.2)
      R=DIST(0.0.0.0.XX,YY)
      TH= ANGL (0.0.0.0.4X, YY)
      CSTH02=305(TH/2.0)
      N1=COS(PHI(I))
      N2=SIN(PHI(I))
      00 250 J=1.4
      XI=0.25*AA*(1.0+XC(J))**2
      RXI=SQRT (R*XI)
      RHO=DIST(XI.O.O.XX.YY)
      RHON=SQRT (R+XI-2.0*RXI*CSTHD2)
      RHOJ=SQPT(R+XI+2.0+RXI+CSTHJ2)
      ALPHA=ANGL (XI.O.O.XX.YY)
      THE TAN=ACOS((SQRT(R) +CSTHD2-SQRT(XI))/RHON)
      THETAD=ACOS((SQRT(R) *CSTHD2+SQRT(X1))/RHOD)
      ARG1=TH/2.0+ALPHA
      ARG2=3.0 + TH/2.0+ALPHA
      ARG3=TH/2.0+2.0*ALPHA
      SEXIDE=SORT(XI/R)
      IF (30TYPE (I.1) . EQ. 1) GO TO 210
```

```
K11=SRXIDR*(SIN(ARG1)+(0.5*YY/R)*COS(ARG2)+(YY/RHO)*COS(ARG3))/RHO
       K12=SRXIDR*((0.5*YY/R)*SIN(ARG2)+(YY/RHO)*SIN(ARG7))/7HD
       K1=-K11*N1-K12*N2
      GO TO 215
210
      K1=-0.25*(KAPPA-1.0)*(THETAN-THETAD)-0.5*SRXIDR*(YY/240)*00S(ARG1)
       T(I) = -T(I)
215
      IF (BOTYPE (I, 2) . EQ. 1) GO TO 220
       K12=SRXIDR*((0.5*YY/R)*SIN(ARG2)+(YY/RH3)*SIN(ARG3))/RH3
      K22=SRXIDR*(SIN(ARG1)-(0.5*YY/R)*COS(ARG2)-(YY/PHO)*COS(ARG3))/RHO
      K2=-K12+N1-K22+N2
      GO TO 245
220
      K2=J.25*(KAPPA+1.0) *ALOG(RHON/RHOD)-0.5*SRXIDR*(YY/RHO) *SIN(ARG1)
      T(I+M) = -T(I+M)
      A(I+M2+J+M4) = (0.5*AA/PI)*K1*(XC(J)+1.0)*W(J)
245
      A(I+M3.J+M4)=(0.5*AA/PI)*K2*(XC(J)+1.0)*W(J)
250
      CONTINUE
      00 265 I=1.N
      A(I+M4.I+M4)=1.0
265
      DO 280 I=1.4
      I1=M2-M2+3DTYPE(I.1)
      I2=M2-M2+BDTYPE(I.2)
      A(I+M2.I+I1)=1.0
      A(I+M3,I+M+I2)=1.0
      00 260 J=1.M2
      B(J,I) = A(J,I+I1)
260
      B(J_{\bullet}I+M) = A(J_{\bullet}I+M+I2)
                                       THIS PAGE IS BEST QUALITY PRACTICABLE
      00 270 J=1.N
                                       FROM COPY FURNISHED TO DDC
      B(J+M4+I)=A(J+M4+I+I1)
270
      B(J+M4,I+M)=A(J+M4,I+M+I2)
280
      CONTINUE
C
C
                    MULTIPLY B(NOEQ.2M) *T(2M)
C
      DO 300 I=1.NOEQ
      RHS(I) =0.0
      DO 300 J=1,42
      RHS(I)=RHS(I)+B(I,J)+T(J)
300
C
C
                    INPUT THE STRESS ON THE CRACK SURFACE
C
      CALL CSTRES(N.XC.SC)
      DO 305 I=1.N
305
      RHS (I+M4) = RHS (I+M4) + SC (I)
C
C
                    SOLVE THE SIMULTANEOUS EQUATIONS AND OUTPUT THE SOLUTION
C
      CALL SIMEQ(A.RHS.CC.II.NDEQ.KO)
      WRITE(6, 1008)
      00 410 I=1.M
      I1=90TYPE(I,1)
      I2=3DTYPE(I.2)
      IF (I1.EQ.O.AND.I2.EQ.O) GO TO 402
         (I1.EQ.O.AND.I2.EQ.1) GO TO 404
      IF
      IF (II.EQ.1.4NO.I2.EQ.0) GO TO 406
      IF (I1.EQ.1.AND.I2.EQ.1) GO TO 408
      WRITE(6, 1011) I,RHS(I),RHS(M+I),RHS(M2+I),RHS(M3+I)
402
      GO TO 410
```

```
WRITE(6,1012) I.-RHS(M+I), RHS(I), RHS(M2+I), -RHS(M3+I)
404
      GO TO 410
406
      WRITE(6,1013) I,-RHS(I),RHS(M+I),RHS(M3+I),-RHS(42+I)
      GO TO 410
438
      WRITE(6,1014) I,-RHS(I),-RHS(M+I),-RHS(M2+I),-RHS(M3+I)
410
      CONTINUE
C
C
                    CALCULATE THE STRESS INTENSITY FACTOR
C
                                    THIS PAGE IS BEST QUALITY PRACTICABLE
      SIF = 0.0
      00 420 I=1.N
                                    FROM COPY FURNISHED TO DDQ
420
      SIF=SIF+RHS(M4+I)*W(I)
      SIF=-SQRT(2.0+AA/PI)+SIF
      WRITE (6, 1015)
      WRITE(6,1016)(I,XC(I),W(I),0.25*AA*(1.0+XC(I))**2,RHS(I+M4),I=1,N)
      WRITE(6,1017) SIF
1000
      FORMAT (2F10.0)
1001
      FORMAT (1H .T40."NO. OF DUTER BOUNDARY SEGMENTS", T89, 15, /, T40.
     1"NO. OF GAUSSIAN INTEGRATION POINTS". T80. 15./. T40.
     2"NO. OF BOUNDARIES", T80, 15,/, T40.
     3"LENGTH OF CRACK". T80. F5. 3./. T40.
     4"MATERIAL PROPERTY - KAPPA". T75.F10.5.//)
1002
      FORMAT (2F20.6)
1006
      FORMAT (//,T22, "SEGMENT", T40, "COORDINATES", T61, "NOPMAL
                                                                    SEGMENT
                BOUNDARY CONDITIONS". / . T33. "BEG". T43. "MID". T53. "END". T71.
     2"LENGTH" . T82, "TYPE" , T94, "X", T104, "Y", //)
1007
      FORMAT (1X. T21. I4. T27. 5F10. 4. T82. 2I2. T90. 2P10. 6. /. T27. 3F10. 4)
      FORMAT(1H .///.T47."DISPLACEMENTS".T92."TRACTIONS/MJ".//.T37."PROB
1008
     1LEM 1". T59, "PROBLEM 2". T83. "PROBLEM 1". T105, "PROBLEM 2"./. T22.
     2"SEGMENT", T35, "X", T46, "Y", T57, "X", T68, "Y", T81, "X", T92, "Y", T103, "X"
     2.T114."Y"./)
1011
      FORMAT (20x,14,5x,22x,2F11.6,2x,2F11.6)
1012
      FORMAT (20X,14,5X,11X,2F11.6,11X,2X,F11.6,22X,F11.6)
1013
      FORMAT (20X,14,5X,F11.6,22X,F11.6,13X,2F11.6)
      FORMAT (20X, 14, 5X, 2F11.5, 46X, 2F11.6)
1014
1015
      FORMAT (///,T43,"NODE
                                            WEIGHT
                                GAUSSIAN
                                                        CRACK
                                                                  STRESS ON
          "./.T50,"COORDINATE".T70,"COORDINATE CRACK"./)
1016
      FORMAT (T40,15, T48, 3F10.6, F12.6)
1017
      FORMAT (/// +T50 +"STRESS INTENSITY FACTOR ="+F10.6+///)
      RETURN
      END
```

FUNCTION SIGN1(X)
SIGN1=1.0
IF (X.LT.0.0) SIGN1=-1.0
RETURN
END

FUNCTION DIST(X1,Y1,X2,Y2)
DIST=SQRT((X2-X1)**2+(Y2-Y1)**2)
RETURN
END

FUNCTION ANGL(X1.Y1.X2.Y2)

ANGL=ATAN2(Y2-Y1.X2-X1)

IF (ANGL.LT.0.0) ANGL=ANGL+6.283185307180

RETURN
END

FUNCTION 0122(C, KAPPA, R, P1, R2)
REAL KAPPA
D122=C*((KAPPA-1.0)*R1-4.0*R1*F2**2)/R
RETURN
END

FUNCTION D222(C,KAPPA,R,R1,R2)
REAL KAPPA
D222=C*((1.0-KAPPA)*R2-4.0*R2**3)/R
RETURN
END

FUNCTION \$122(C,KAPPA,R,R1,R2,N1,N2)

REAL KAPPA,N1,N2

\$122=4.0*C*((-1.0+8.0*R1**2*R2**2)*N1+(-2.0*R1*R2+3.)*R1*R2**3)

1*N2)/R**2

RETURN

END